

An Analytical Method for Optimum Frequency Domain Design in Control Applications

Gary L. Dempsey, Eugene S. McVey
Bradley University/University of Virginia

Abstract

Our paper will revisit a method that was first introduced at Yale University in the late 1950s by Bower and Schultheiss in their linear control textbook, Introduction to the Design of Servomechanisms. Control system design normally requires several design iterations in a simulation environment to meet a set of system specifications. The analytical method discussed here comprises the first design iteration but normally results in a design that is close to optimum, i.e., the resulting system will have maximum stability margin and bandwidth. The method eliminates trial and error procedures, can accommodate time and frequency domain specifications, and can be applied to continuous-time or sampled-data systems.

I. Introduction

Undergraduate control theory textbooks and courses have typically been heavily weighted to control system analysis while design approaches have received less attention. Design approaches are many times reduced to cookbook methods that are not optimum for a given application. The cookbook methods and lack of design examples do not provide insight into how to modify the controller if a set of system specifications is not met. Therefore, students are unable to solve problems they have not seen in the course or textbook or rely on trial and error procedures to meet specifications. Obviously, analytical design methods for the controllers would be beneficial to undergraduate engineering students and to the control theory community.

Our paper will revisit a method that was first introduced at Yale University in the late 1950s by Bower and Schultheiss¹. The method was illustrated for several minimum-phase continuous-time systems and a system modeled as first-order with time delay. During this period Dr. J. E. Gibson taught the method in control theory courses at Purdue University. The method was observed later in other textbooks although not in its entirety²⁻⁴. The authors have presented the method in their control theory courses at Bradley University, Purdue University and the University of Virginia.

The paper is divided into the following five sections. Section II will provide motivation for the analytical method proposed by Bower and Schultheiss. Section III will introduce the frequency domain design method for minimum-phase continuous-time systems. Section IV will show how the method can

accommodate time and frequency domain specifications. Section V will expand the method to non-minimum phase and sampled-data systems. Section VI will briefly discuss the undergraduate control theory sequence at Bradley University for electrical and computer engineering students.

II. Frequency Domain Design

Most real world systems are better described in the frequency domain. For example, time delay can be treated exactly in the frequency domain. Time domain analysis and design (root locus) is normally the first approach to present in control theory classes. We feel that time domain methods and the associated specifications should be clearly understood before teaching frequency domain approaches. Some textbooks do a better job showing the design procedures and the iterative process than others. A good example is the textbook by Rohrs, Melsa, and Schultz⁴. One design example shown by the authors uses the optimum phase margin method described in the next section.

In undergraduate and introductory graduate textbooks that cover frequency domain design, authors concentrate on shaping the magnitude versus frequency curve using gain, lag, lead, lag-lead, and PID-type compensators to meet a set of specifications. Little or no attention is given to shaping the phase versus frequency curve. There are a number of textbooks that cover shaping the magnitude curve in detail^{1,3-8}. Shaping of the phase curve will be the focus of this paper although the techniques for magnitude shaping must be understood as well.

The open-loop frequency response of a minimum-phase system $G(s)$ in the vicinity of the crossover frequency ω_c is shown in Fig. 1. It is always desired to force the magnitude curve's slope at crossover to be -1 (log-log plot) or -20 db per decade (semi-log plot). This will yield a stable system with a phase margin between 0 and 90 degrees which depends on the length of the -1 segment. The ideal situation for the shape of the phase curve is shown in Fig. 1. The peak of the phase curve occurs at ω_c thus yielding the maximum phase margin and creating a system that is robust to plant variations. A given phase margin (PM) specification will determine the length of the -1 slope segment.

A more important specification to the user after stability is assured is system speed (a time domain specification). Maximizing ω_c will maximize closed-loop bandwidth and speed. Relationships and equations between the time and frequency domains will be covered in Section IV. The objective of the optimum PM margin design method is to shape the phase curve in the form of a peak at ω_c and to increase ω_c as far as the constraints of the system will allow. Control system design normally requires several design iterations in a simulation environment to meet a set of system specifications. The optimum PM method comprises the first design iteration but normally results in a design that is close to optimum, i.e., the resulting system will have maximum stability margin and bandwidth.

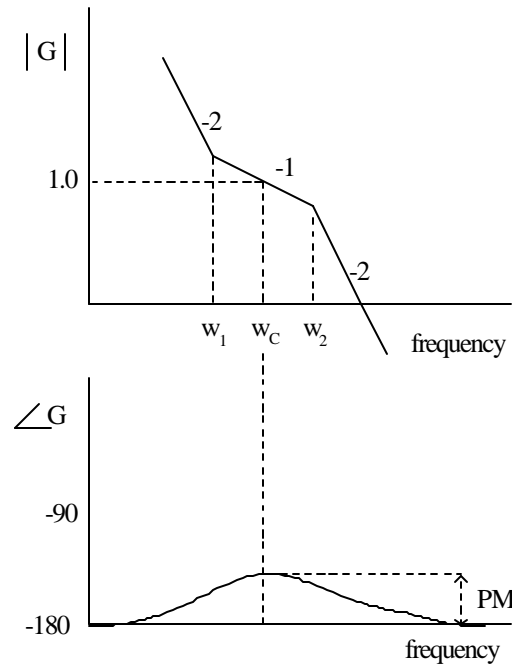


Fig. 1. Open-Loop Frequency Response Example.

III. Optimum Phase Margin Design

Phase margin is shown graphically in Fig. 1 or can be stated as

$$PM = 180^\circ + \angle G(jw_c) = 180^\circ + B_c \quad (1)$$

The phase angle B_c of the open-loop transfer function $G(s)$ at w_c is

$$B_c = -p + \tan^{-1} \frac{w_c}{w_1} - \tan^{-1} \frac{w_c}{w_2} \quad (2)$$

assuming the -2 segments are infinite length. Equation (2) can be simplified for design purposes as

$$B_c \approx -p + \left(\frac{p}{2} - \frac{w_1}{w_c}\right) - \frac{w_c}{w_2} \quad (3)$$

using approximations for the arc tan function. Expressing the arc tan function as a power series

$$\tan^{-1} x = x - \frac{x^3}{3} + \frac{x^5}{5} - \dots \approx x \quad \text{for } x < 1 \quad (4)$$

$$\tan^{-1} x = \frac{p}{2} - \frac{1}{x} + \frac{1}{3x^3} - \frac{1}{5x^5} + \dots \approx \frac{p}{2} - \frac{1}{x} \quad \text{for } x > 1 \quad (5)$$

shows that eliminating the high-order terms introduces minimal error because of the alternating

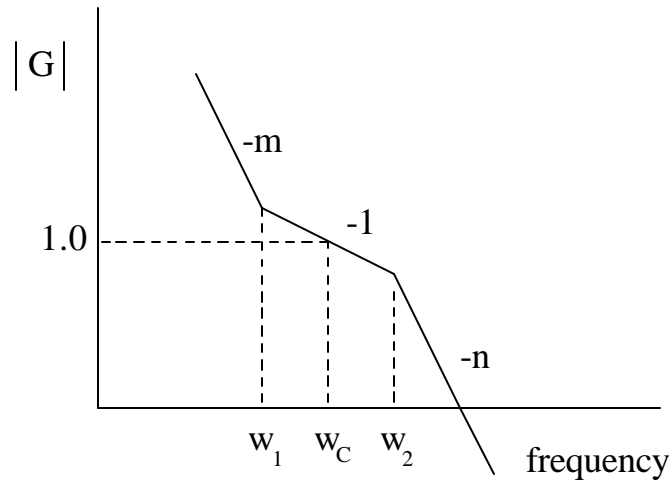


Fig. 2. General Shape in the Vicinity of the Crossover Frequency.

series and the first nonlinear term is cubic. The optimum w_c in Fig. 1 can be determined by differentiating (3) with respect to w_c

$$\frac{dB_c}{dw_c} = \frac{w_1}{w_c^2} - \frac{1}{w_2} = 0 \quad \text{or} \quad w_c = \sqrt{w_1 w_2} \quad (6)$$

Refer to Fig. 2 for derivation of optimum w_c for general shapes in the vicinity of w_c . The phase angle of the open-loop transfer function $G(s)$ at w_c is

$$B_c = -\frac{p}{2}m + (m-1)\left(\frac{p}{2} - \frac{w_1}{w_c}\right) - (n-1)\frac{w_c}{w_2} \quad (7)$$

assuming infinite m and n segments. The optimum w_c for the general shapes is

$$w_c = \sqrt{\frac{m-1}{n-1} w_1 w_2} \quad (8)$$

Consider the continuous-time system shown in Fig. 3 for an example of applying the optimum PM approach. The plant is

$$Gp(s) = \frac{10}{(0.1s+1)(0.0005s+1)} \quad (9)$$

The system specifications are:

1. steady-state error (e_{ss}) to step inputs=0
2. steady-state error (e_{ss}) to ramp inputs<0.1%
3. phase margin ≥ 50 degrees

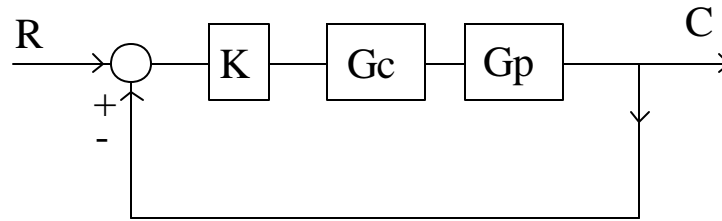


Fig. 3. Continuous-Time Example.

4. minimize rise time (t_r) for step inputs
5. noise level (open-loop) $\leq 10\%$ at frequencies ≥ 1000 rad/sec

The steady-state error specification for step inputs requires the compensator $G_c(s)$ to have one integration. The ramp error specification is used to determine the proportional gain K as follows:

$$e_{ss} = \frac{1}{K_v} = 0.001 \text{ or } K_v = 1000 \quad (10)$$

where

$$K_v = \lim_{s \rightarrow 0} sKG_c(s)G_p(s) = 10K = 1000 \quad (11)$$

Therefore the proportional gain K is equal to 100 to just meet the specification. The partially compensated open-loop transfer function is

$$KG_c(s)G_p(s) = 100 \left(\frac{1}{s} \right) \left(\frac{10}{(0.1s + 1)(0.0005s + 1)} \right) \quad (12)$$

The magnitude versus frequency sketch is shown in Fig. 4 (violet-blue curve). Our students are required to perform the first design iteration on graph paper before using a simulation tool. From Fig. 4, ω_c is 100 rad/sec. Using the arc tan approximations, the phase angle at ω_c is

$$B_c \approx -\frac{p}{2} - \left(\frac{p}{2} - \frac{10}{100} \right) - \frac{100}{2000} \approx -3.09 \text{ rad} \approx -177^\circ \quad (13)$$

Using (1), the PM is 3 degrees (almost unstable).

It is desired to crossover with a -1 segment to maximize PM. To minimize rise time (maximize speed in the time domain), a PM of 50 degrees is used. Higher PM results in a slower system (longer rise time). A lead network can produce the desired shape at ω_c and achieve a larger ω_c than the partially compensated system shown in Fig. 4. Refer to Fig. 4 for a quick sketch of the proposed compensated open-loop system (violet-red curve). Note that there are three unknowns, ω_1 , ω_c , and ω_2 , where ω_1 and ω_2 will be determined by the lead network. For optimum PM,

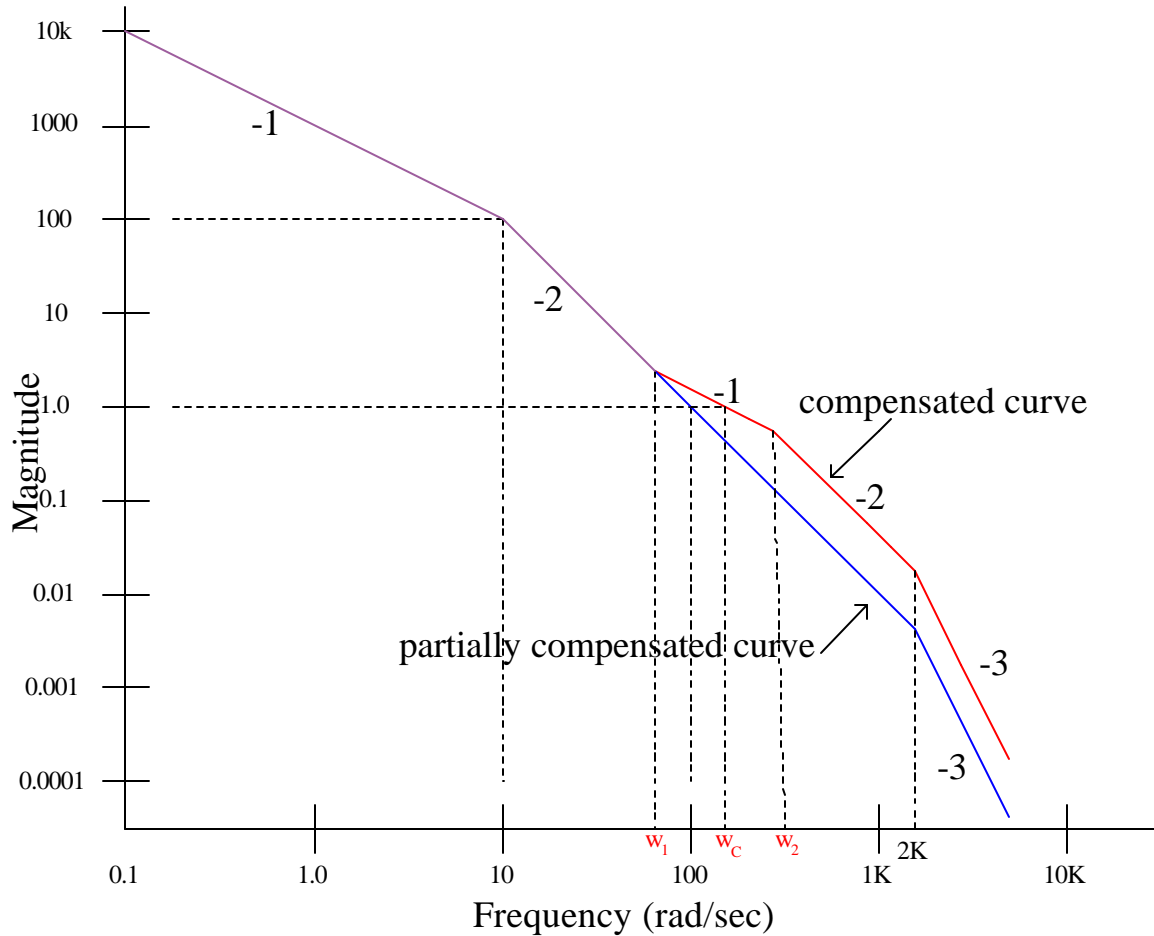


Fig. 4. Quick Sketch of Compensated Open-Loop Magnitude Curve.

$$w_c = \sqrt{w_1 w_2} \quad \text{or} \quad w_1 = \frac{w_c^2}{w_2} \quad (14)$$

The desired angle B_c for a PM of 50 degrees is

$$B_c = -130^\circ = -2.27 \text{ rad} = -p + \left(\frac{p}{2} - \frac{w_1}{w_c} \right) - \frac{w_c}{w_2} \quad (15)$$

assuming infinite length segments for the -2 slopes. Substituting w_1 from (14), (15) is reduced to

$$-2.27 \text{ rad} = -\frac{p}{2} - 2 \frac{w_c}{w_2} \quad (16)$$

It is desired to maximize w_c but there are a number of practical constraints. Assuring a reasonable

reduction in noise at high frequencies will limit w_c . Also, the high frequency dynamics of the plant may restrict crossover frequencies. For this example, the noise level specification requires the magnitude of the open-loop transfer function to be ≤ 0.1 for frequencies ≥ 1000 rad/sec. From a quick sketch of the compensated magnitude curve, w_c should be selected to be less than 200 rad/sec. For the first design iteration let w_c equal 150 rad/sec. From (16), w_2 is equal to 429 rad/sec and from (14), w_1 is equal to 52 rad/sec. The final first iteration compensator is

$$KGc(s) = \left(\frac{100}{s} \right) \frac{s/52 + 1}{s/429 + 1} \quad (17)$$

This compensator is used to generate the exact magnitude-phase versus frequency curves using the Control System Toolbox from The Math Works Inc. The exact PM is 48.9 degrees with a w_c of 183 rad/sec. However, the design is not optimum. The peak of the phase curve is at 110 rad/sec. Reduction of the proportional gain K from 100 to 54 will align w_c with the peak of the phase curve. This will yield a PM of 52 degrees.

Unfortunately more design iterations are required because the gain reduction causes the ramp steady-state error to fall out of specification. Experience will lead to minimizing the number of design iterations. A better approach would be to push the peak of the phase curve toward the present crossover frequency. After three more iterations using the Control System Toolbox, the final compensator is

$$KGc(s) = \left(\frac{100}{s} \right) \frac{s/65 + 1}{s/700 + 1} \quad (18)$$

The final compensated open-loop frequency response is shown in Fig. 5. The phase margin is 54 degrees at a crossover frequency of 161 rad/sec. A further increase in w_c would violate the noise level specification. After the first design iteration is performed on graph paper, we like to refer to subsequent iterations as “educated trial and error” with a simulation package. As stated from the Bower and Schultheiss textbook, "It is not our aim to compile a handbook listing all possible contingencies". We agree with their statement although we have provided numerous continuous-time and sampled-data system examples to our students.

IV. Control Specifications

The optimum PM approach may require equations that relate time domain or closed-loop frequency domain specifications back to the open-loop parameters PM and crossover frequency. The majority of undergraduate control theory textbooks cover these relationships³⁻⁸. As an example, one of the more

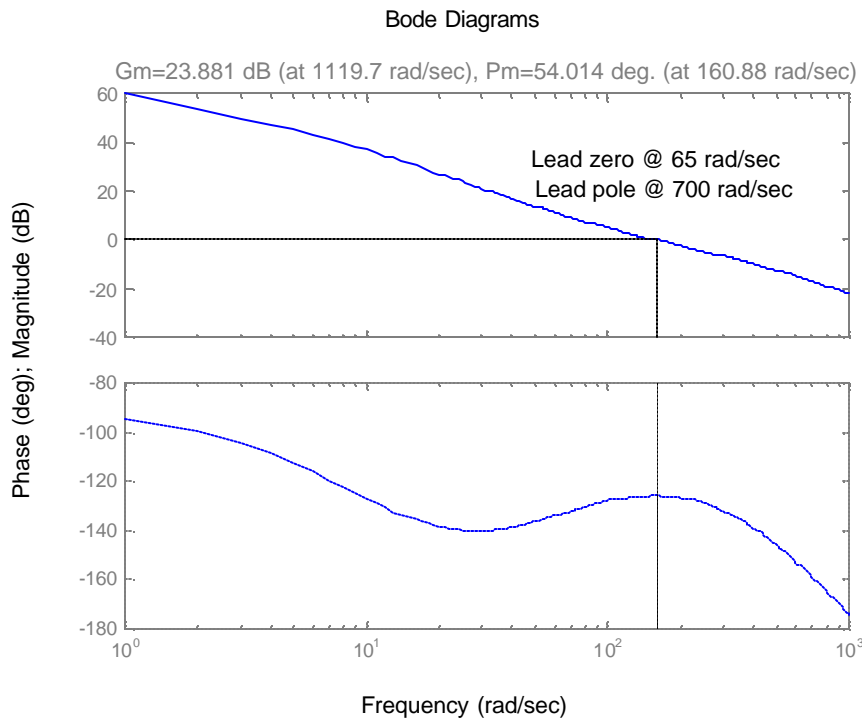


Fig. 5. Final Compensated Open-Loop Frequency Response.

important time domain specifications is percent overshoot for step inputs. In terms of the damping ratio d , percent overshoot is

$$\% O.S. = 100e^{\frac{-pd}{\sqrt{1-d^2}}} \quad (19)$$

An equation to tie this specification back to the open-loop frequency domain is

$$PM \approx 100d \quad \text{for } PM < 70^\circ \quad (20)$$

A critical parameter in the closed-loop frequency domain is M_p (maximum peak value). The relationship between M_p and PM is

$$\sin(PM) \approx (M_p)^{-1} \quad (21)$$

Normally the user can expect more design iterations when the specifications are given in the time domain or closed-loop frequency domain because of the approximation equations. Also the specification equations are derived for an exact second-order system⁶.

V. Nonminimum-Phase and Sampled-Data Systems

For minimum-phase systems, phase can be determined indirectly from the magnitude versus frequency

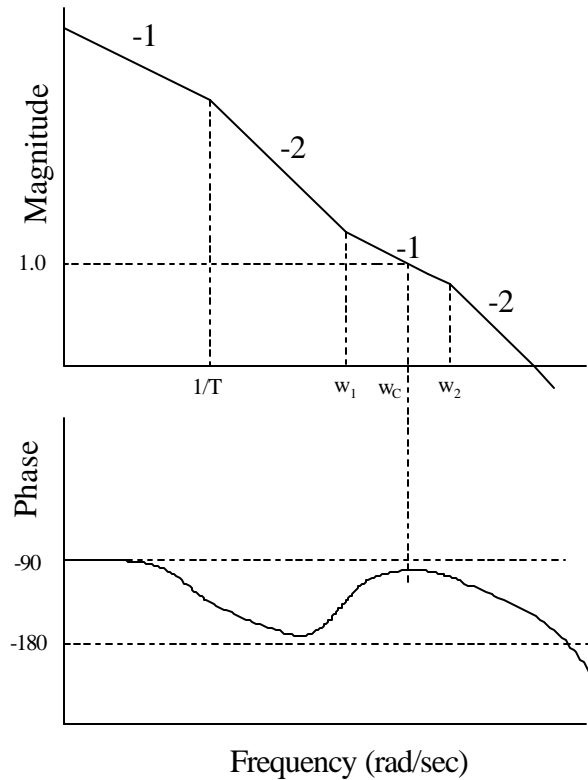


Fig. 6. Nonminimum-Phase System Compensation Example.

curve. Nonminimum-phase and sampled-data systems require using both magnitude and phase versus frequency curves. Consider the system shown in Fig. 3 with a first-order plant with time delay T_D ,

$$Gp = \frac{Ke^{-sT_D}}{Ts + 1} \quad (22)$$

A suggested compensation method is shown in Fig. 6 where

$$Gc = K_p \frac{\frac{s}{z} + 1}{s \left(\frac{s}{p} + 1 \right)} \quad (23)$$

The steady-state error and system speed is improved with the PID-type compensation. The angle at crossover is

$$B_c \approx -p + \left(\frac{p}{2} - \frac{w_1}{w_c} \right) - \frac{w_c}{w_2} - w_c T_D \quad (24)$$

Differentiating (24) with respect to w_c and solving for w_c for the optimum PM yields

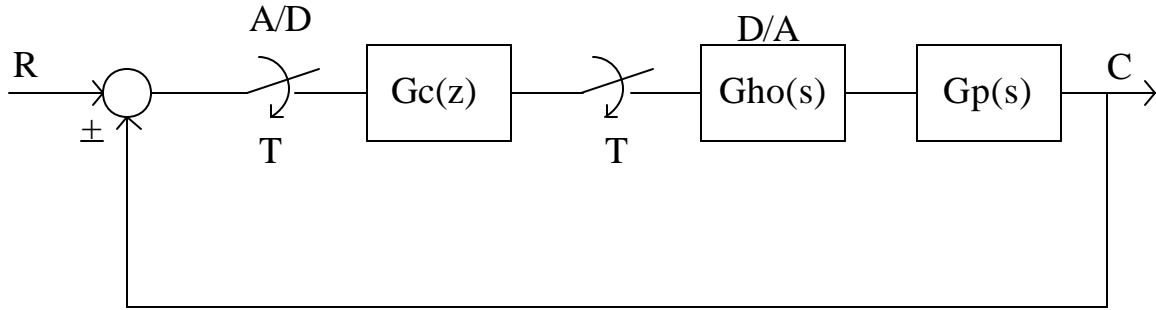


Fig. 7. Sampled-Data System Example.

$$w_c = \sqrt{\frac{w_1}{T_D + 1/w_2}} \quad (25)$$

The suggested compensation method in Fig. 6 may not be practical for systems with a large time delay component at w_c . The lead portion of the compensator is used to cancel the effect of the time delay's phase lag at w_c . However, large time delays may require a long -1 segment which may not be feasible for some systems.

Consider the sampled-data system shown in Fig. 7. In this application, the compensation is implemented in software ($G_c(z)$). A digital-to-analog converter (G_{ho}) is used to drive the continuous-time plant $G_p(s)$. The A/D converter is modeled as an ideal sampler.

Method 1: A continuous-time approach can be used to determine $G_c(s)$ using an approximate model for the zero-order hold device. Once $G_c(s)$ is obtained, $G_c(z)$ can be found using one of the common discretization methods such as the bilinear transformation⁹. In the frequency domain, the zero-order hold is described as

$$G_{ho}(j\omega) = T \frac{\sin\left(\frac{p\omega}{w_s}\right)}{\frac{p\omega}{w_s}} \quad \text{with angle} = -p \frac{\omega}{w_s} \quad (26)$$

where w_s is the sampling frequency. For a sampling rate $w_s \geq 6w_c$,

$$G_{ho} \approx 1.0 \quad \text{with an angle} = -p \frac{\omega}{w_s} \quad (27)$$

The B_c equation for the system requires a simple modification of adding the phase lag term for the

zero-order hold. The design procedure is similar to the method discussed previously for a system with time delay.

Method 2: Design in the w -plane is an alternative method for determining $G_c(z)$ in Fig. 7. The exact representation of the zero-order hold is used in this method. A right-half plane zero is produced in the w -plane due to the zero-order hold. The design method is discussed in Ogata's textbook⁹. The optimum PM design in the w -plane is similar to the methods shown previously. Once $G_c(w)$ is determined, the inverse transformation is used to describe $G_c(z)$.

VI. Undergraduate Control Sequence at Bradley University and Concluding Remarks

The first control theory course is offered fall semester of the senior year. This classical control course is divided into three equal sections: modeling, time domain analysis and design (root locus), and frequency domain analysis and design. Gain, lag, lead, lag-lead, and PID-type controllers are introduced with the root locus design method. Only gain compensation is discussed with the frequency domain approach because of time constraints but the type of plants are open-ended including plants with time delay.

The second control theory course (spring semester) is divided into three sections: continuation of frequency domain design (20%), digital control (40%), and the state-variable control method (40%). The course begins with frequency domain compensation (shaping) to meet a set of specifications for continuous-time systems. Gain, lag, lead, lag-lead, and PID-type controllers are introduced as well as the optimum PM approach. Frequency domain and optimum PM are again the primary design methods for the digital control section of the course. The last 40% of the course introduces state-variable modeling, analysis, and design. Note that the emphasis in the two course sequence is classical control. The first graduate control course is devoted to the state-variable method.

In the undergraduate classes, the optimum frequency domain design methods are applied to several types of analog systems including ones with time delay and also sampled-data systems. Students are required to perform anywhere from 2 to 5 design iterations in homework and take-home exam problems. Because of the different systems (electronic, thermal, hydraulic, mechanical) and types of signals (analog, digital, or mixture), new equations must be derived to account for these differences. These equations are applied in the first design step. The next design step involves use of control system simulation software. The Control System Toolbox, MATLAB, and SIMULINK has been the platform used at Bradley. Once the control system is optimized in simulation, the controller is then converted to software or hardware for implementation.

The optimum phase margin approach has been used in senior capstone projects which have consisted of phase-locked loops, switching power supplies, and a temperature controller. It is currently being applied to a small robot arm system by two senior students and to an industrial hydraulic application by a

graduate student for her thesis. Although the method requires a mathematical description of the plant, a high precision model is not required. It is especially useful in electronic control applications where the model can usually be determined directly from a data sheet. Examples are switching power supply integrated circuits and phase-locked loop circuits.

Bibliography

1. Bower, J. L. and Schultheiss, P. M. Introduction to the Design of Servomechanisms. John Wiley & Sons, (1958).
2. Eveleigh, V.W. Adaptive Control and Optimization Techniques. McGraw-Hill Book Company (1967).
3. Eveleigh, V.W. Introduction to Control Systems Design. McGraw-Hill Book Company (1972).
4. Rohrs, C. E., Melsa, J. L., and Schultz, D. G. Linear Control Systems. McGraw-Hill, Inc. (1993).
5. D'Azzo, J. J. and Houpis, C. H. Linear Control System Analysis and Design. McGraw-Hill, Inc. (1995).
6. Franklin, G. F., Powell, J. D., and Emami-Naeini, A. Feedback Control of Dynamic Systems. Addison-Wesley Publishing Company (1991).
7. Kuo, B. C. Automatic Control Systems. Prentice Hall (1991).
8. Ogata, K. Modern Control Engineering. Prentice Hall (1997).
9. Ogata, K. Discrete-Time Control Systems. Prentice Hall (1987).

GARY L. DEMPSEY

Gary Dempsey received the Ph.D. in Electrical Engineering from the University of Virginia in 1991. Dr. Eugene McVey served as dissertation advisor. He is currently an Associate Professor in the Electrical and Computer Engineering Department at Bradley University in Peoria, IL. He currently teaches the undergraduate control theory sequence, senior and graduate laboratories, and artificial neural networks.

EUGENE S. McVEY

Eugene McVey received the Ph.D. degree in engineering from Purdue University in 1960. He was Instructor and Assistant Professor from 1957 to 1961 at Purdue University. From 1962 to 1994, he was Associate Professor (1961-1966) and Professor (1966-1994) at the University of Virginia. He is the author of over 140 archival publications, holds 18 patents, and supervised 30 Ph.D. dissertations and 55 Master Theses during his teaching career.

Obviously, analytical design methods for the controllers would be beneficial to undergraduate engineering students and to the control theory community. Our paper will revisit a method that was first introduced at Yale University in the late 1950s by Bower and Schultheiss¹. The method was illustrated for several minimum-phase continuous-time systems and a system modeled as first-order with time delay. During this period Dr. J. E. Gibson taught the method in control theory courses at Purdue University. McVey, E., & Dempsey, G. (2001, June), An Analytical Method For Optimum Frequency Domain Design In Control Applications Paper presented at 2001 Annual Conference, Albuquerque, New Mexico. 10.18260/1-2--8904. Download Citation. —. Parameter optimized controller design based on frequency domain identification. A thesis submitted to the department of electrical and. Electronics engineering and the institute of engineering and sciences. bS3l, 26. I certify that I have read this thesis and that in my opinion it is fully adequate, in scope and in quality, as a thesis for the degree of Master of Science. Assoc. Prof. Dr. Omer McJrgii^Supervisor). I certify that I have read this thesis and that in my opinion it is fully adequate, in scope and in quality, as a thesis for the degree of Master of Science. Prof. Dr. Erol Sezer. I certify that I have read this thesis and that in my opinion it is fully adequate, in scope and in quality, as a thesis for the degree of Master of Science. Frequency Response Analysis & Design. In conventional control-system analysis there are two basic methods for predicting and adjusting a system's performance without resorting to the solution of the system's differential equation. They are: Root-Locus Method. Frequency-Response Method. For the comprehensive study of a system by conventional methods it is necessary to use both methods of analysis. Mechatronics Frequency Response Analysis & Design. In designing a closed-loop system, we adjust the frequency-response characteristic of the open-loop transfer function by using several design criteria in order to obtain acceptable transient-response characteristics for the system. Mechatronics Frequency Response Analysis & Design. K. Craig 4. Part of the Controls and Control Theory Commons Recommended Citation Kealy, T., O'Dwyer, A.: Analytical ISE Calculation And Optimum Control System Design. Irish Signals and Systems Conference, University of Limerick, Ireland, 2003. This Conference Paper is brought to you for free and open access by the School of Electrical and Electronic Engineering at ARROW@DIT. It has been accepted for inclusion in Conference papers by an authorized administrator of ARROW@DIT. For more information, please contact yvonne.desmond@dit.ie, arrow.admin@dit.ie. The asymptotic stability of the closed-loop system is a basic requirement when searching for optimal controller settings.